

Impact of Energy-Efficiency Parameters on Home Humidity

Rob Vieira Florida Solar Energy Center

A Research Institute of the University of Central Florida







- Some energy efficiency measures only address sensible loads. As these measures are implemented
 - Can today's HVAC equipment maintain acceptable humidity levels?
- Are there some key efficiency measures that could improve humidity in homes that implement sensible load reduction strategies?





- Sensible: HeatLatort: Maistur
- Latent: Moisture





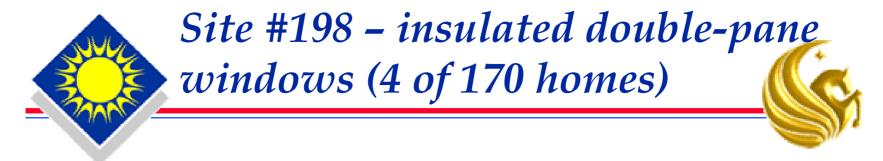
- Single-family detached homes
 - Multi-family residences can often have less envelope sensible load due to fewer exposed surfaces but have not been the focus of many research studies – they are not covered in this presentation
- Existing studies that focus on humidity levels in homes





- Efficient Windows (experimental & simulated)
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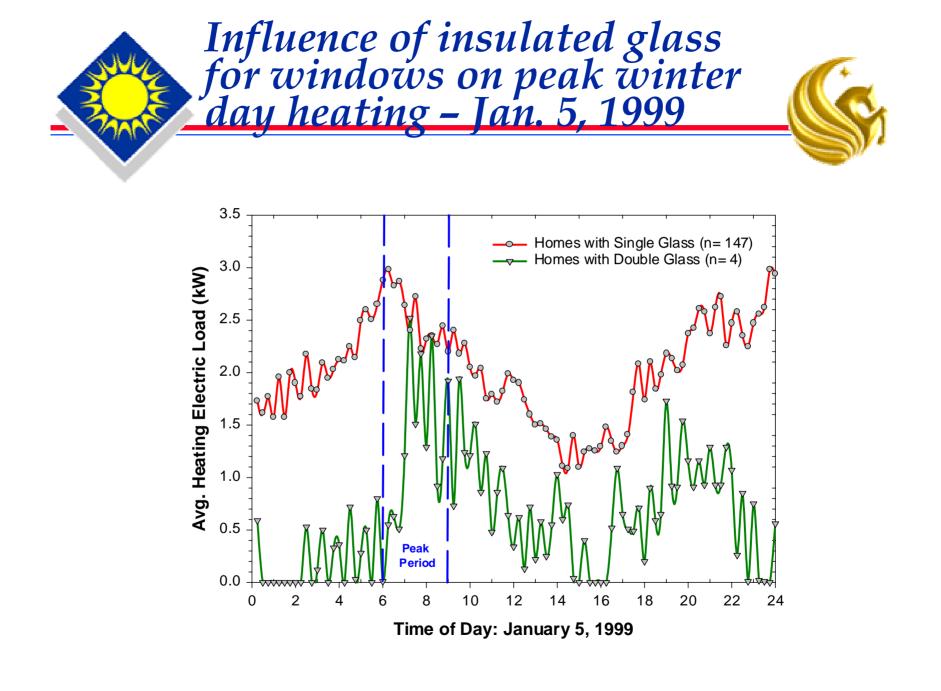


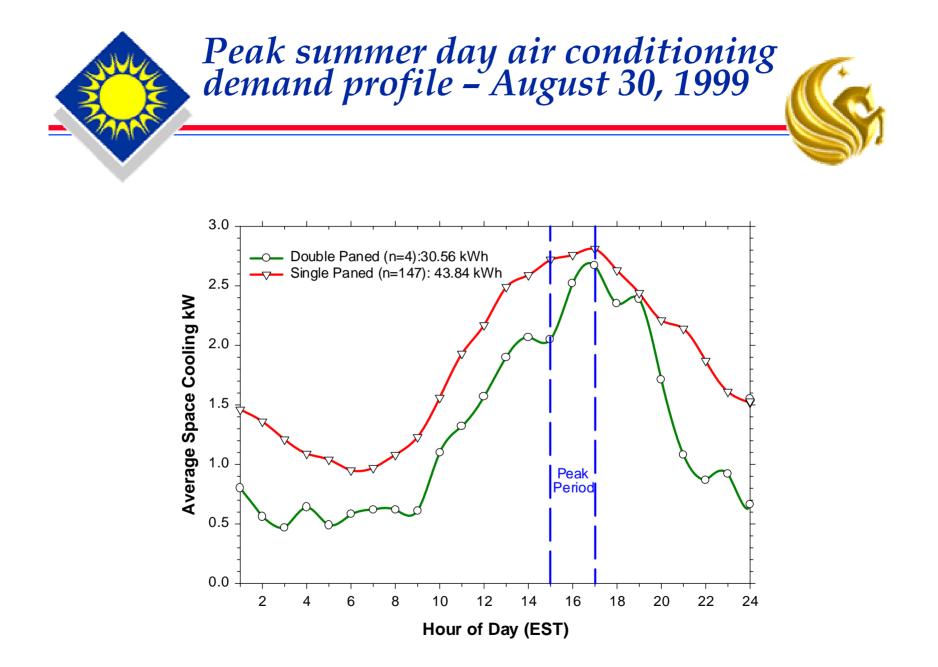




Case	Annual Heating	Annual Cooling	Peak Heating	Peak Cooling
Single Glass	1067	5754	2.60	2.93
Double Glass	485	3318	1.85	2.77

Note: Homes with all double glass: sites 23, 32, 110 and 198. There were 166 homes in the single glass sample.







- Monitoring can answer direct and indirect questions
- What will they save?
 - Energy
 - Peak demand
- How will it impact interior humidity?



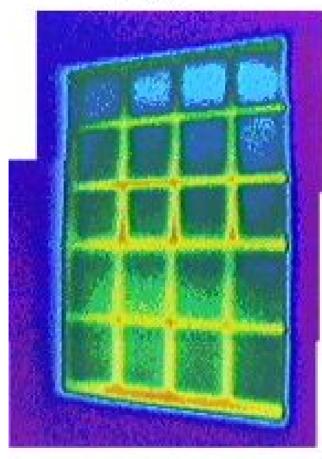
Base and Improved Window Specs Mercedes Homes Research Project



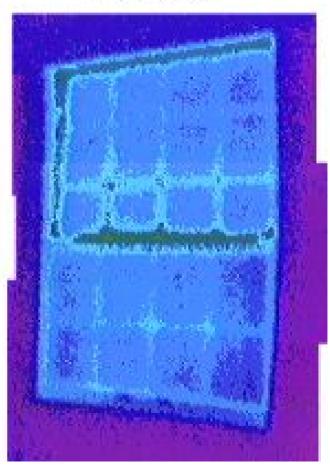
	Solar Heat Gain Coefficient	Winter Night U-Factor
Standard Single Pane Aluminum Windows	0.77	1.23
High Performance Windows - Spectrally Selective, Thermally broken	0.36	0.47



Base

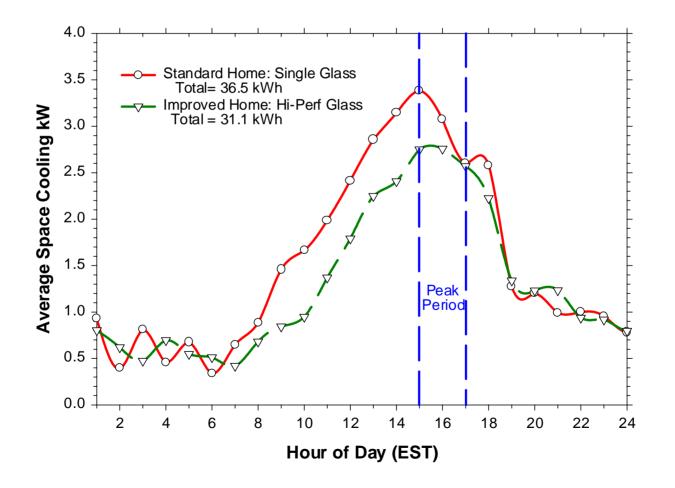


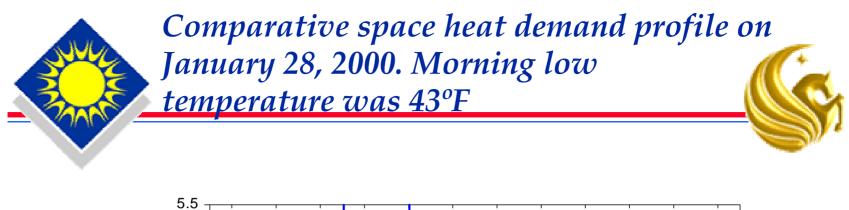
Improved

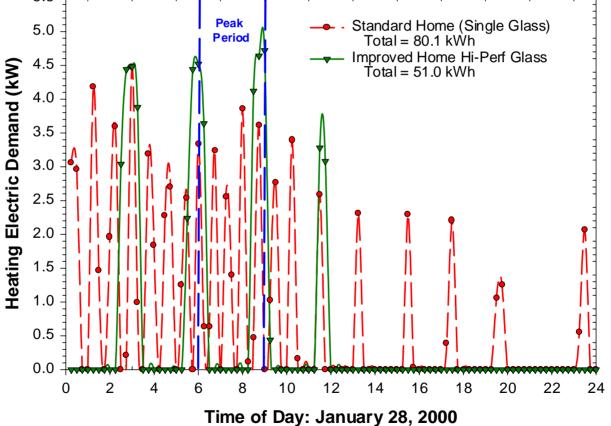




Comparative space cooling demand profile of two test homes over the hot 17-day period in Sept.

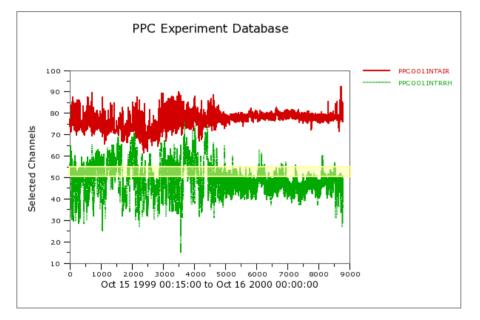




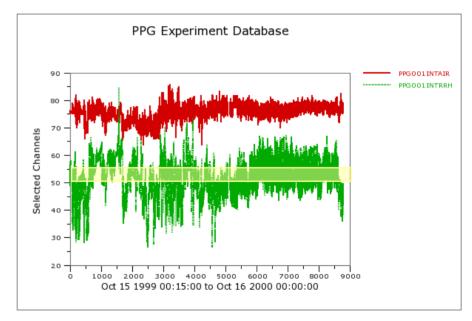




Control

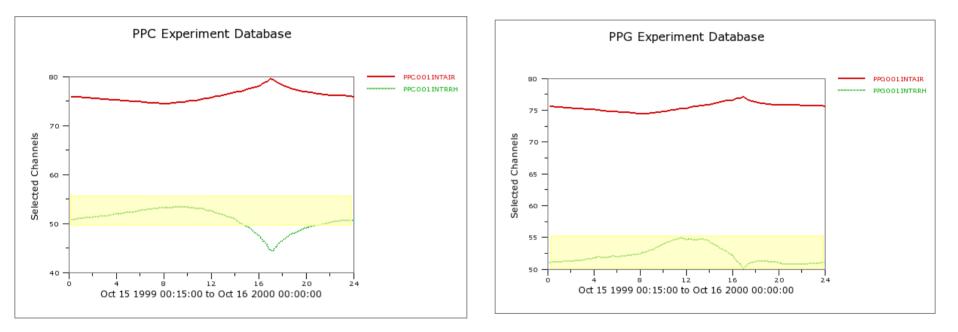


Experimental





Control, Single Glass: Avg Temp= 76.0 F, RH= 50.7%

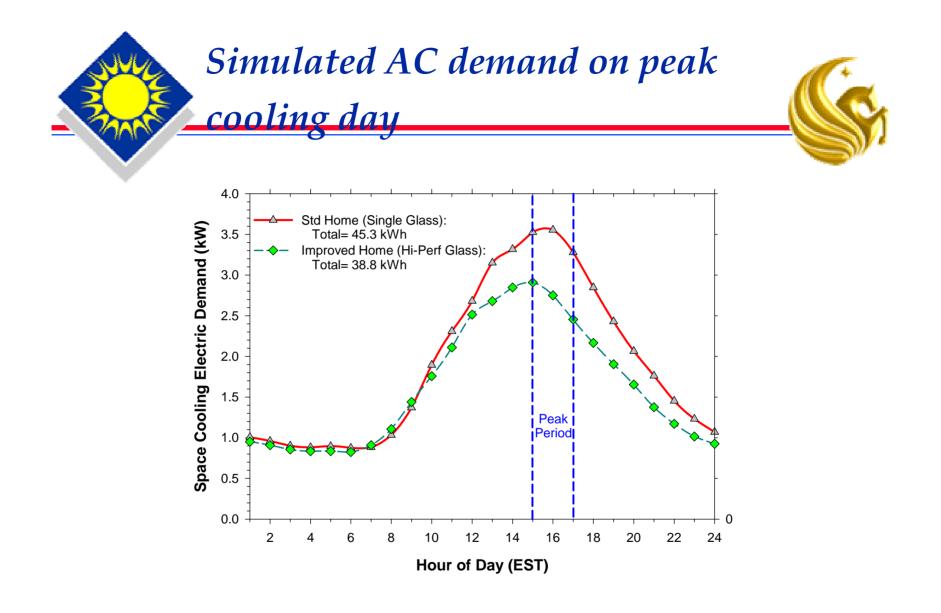


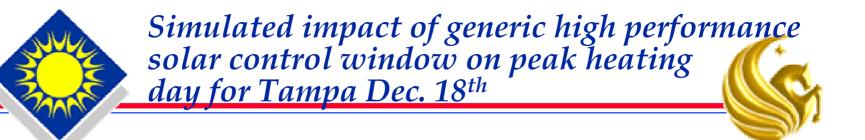
Solar Control Glass: Avg Temp= 75.5 F, RH= 52.1%

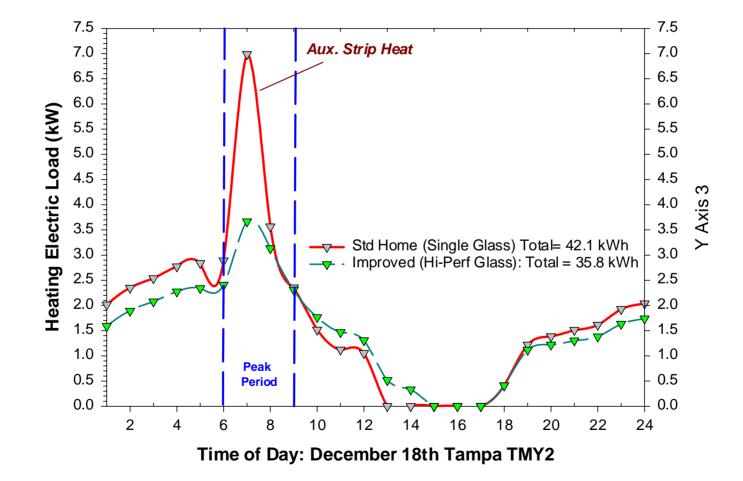
Simulated Impact of Solar Control Windows

- ✤ 75 F set point in summer
- ✤ 72 F set point winter
- No change in AC size
- Annual average relative humidity
 - 41.6% Single Glass
 - 43.7% Solar Control Glass
 - > 2% increase in interior RH
 - Similar to empirical study













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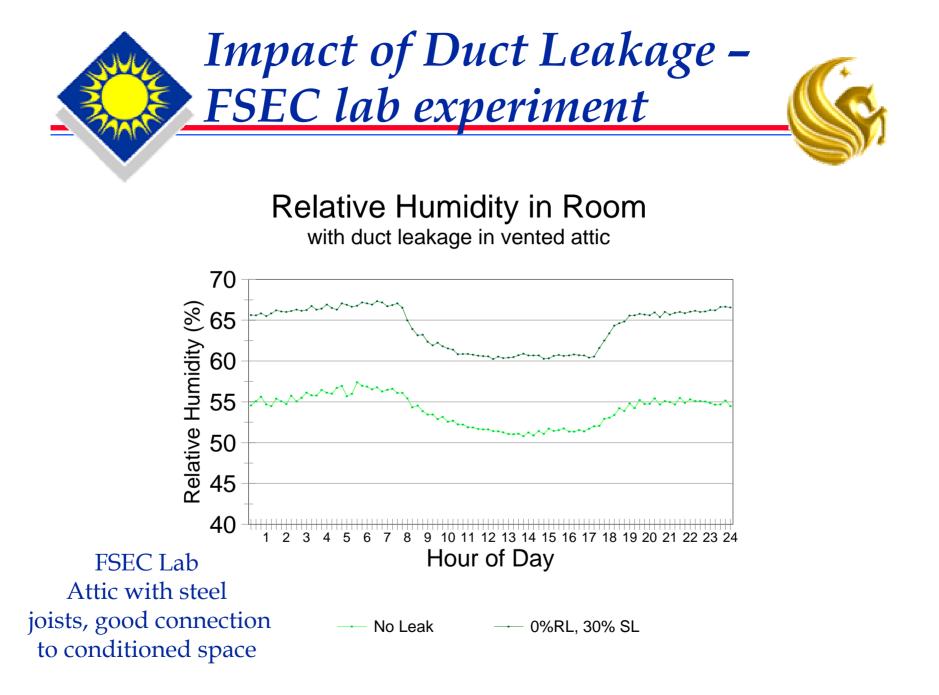
*	Case	CoolingkWh	HeatingkWh	Avg RH%
	Standard Building:	5958	585	48.4%
*	Low-E solar windows	s 4213	292	51.3%
*	White Roof	5186	595	49.3%
*	100% CFL Lighting	5602	625	48.7%
*	Reduce Infiltration	5517	481	46.1%

FSEC EnergyGauge USA analysis, 2007





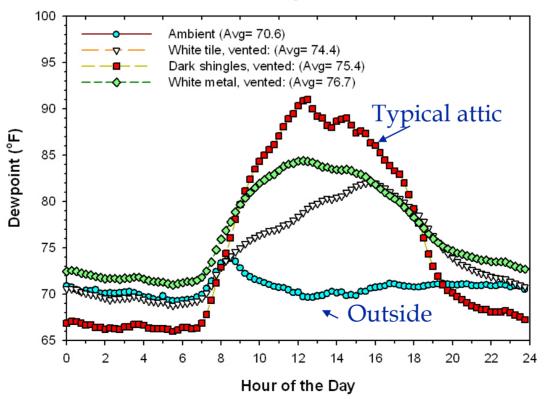
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Measured Average Mid-attic Dewpoints FRF: August 2000





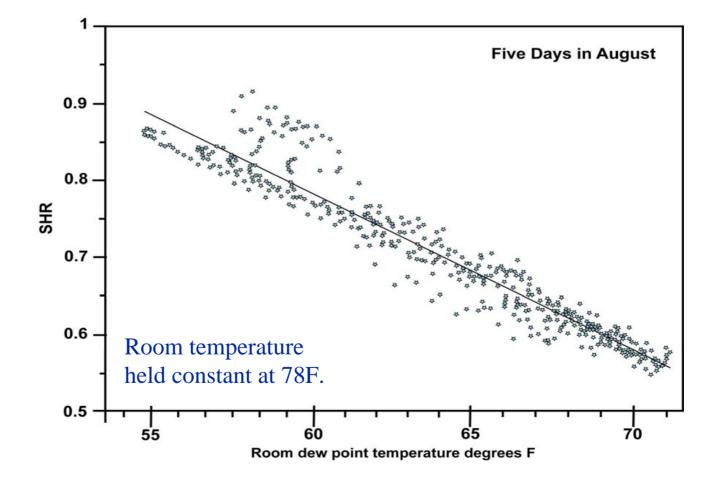


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- ✤ Rated sensible heat ratio (SHR) is about 75%
- ✤ Latent heat ratio (LHR) is about 25%
- However, LHR increases
 - With higher room humidity
 - > With lower system air flow rates

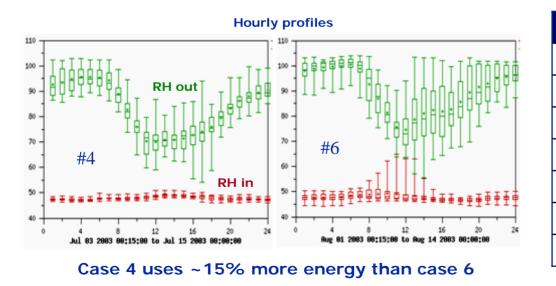


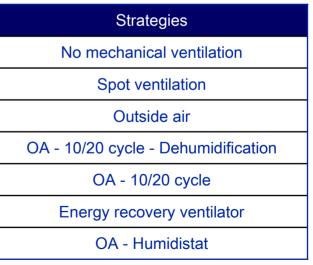




- Automatic simulation of sensible and latent occupancy loads.
- 6 ventilation strategies
- Cooling set point 75F, A/C adequate to maintain RH below 60% for all strategies examined.







Presented paper: "Assessing Six Residential Ventilation Techniques in Hot and Humid Climates," Proceedings of ACEEE 2004 Summer Study on Energy Efficiency in Buildings, Washington, DC, August 2004. .



- "We conclude that the latent removal of small unitary equipment is not a function of efficiency.
- Furthermore, the latent removal characteristics of this type of equipment have not changed significantly over time..."
- David Godwin, "Latent Capacity of Unitary Equipment," ASHRAE TO-98-9-2



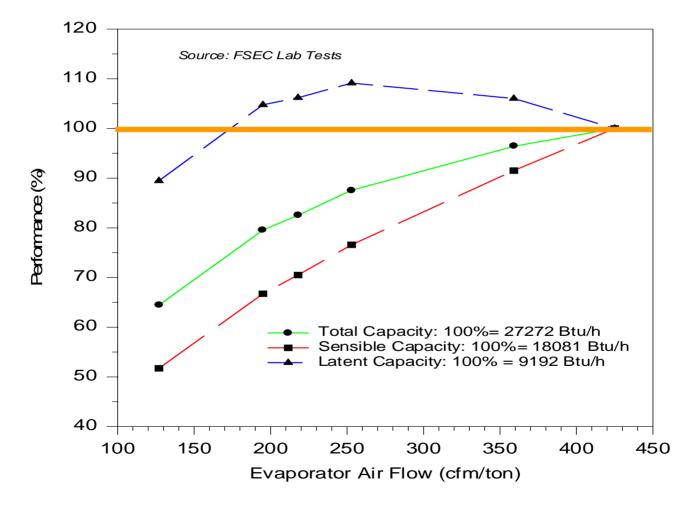


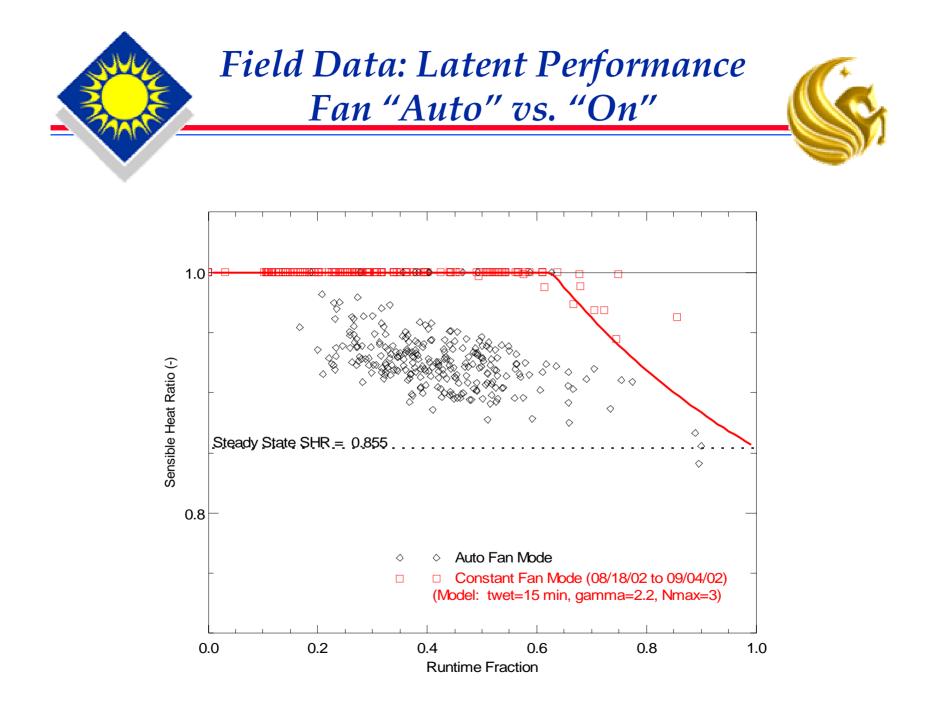
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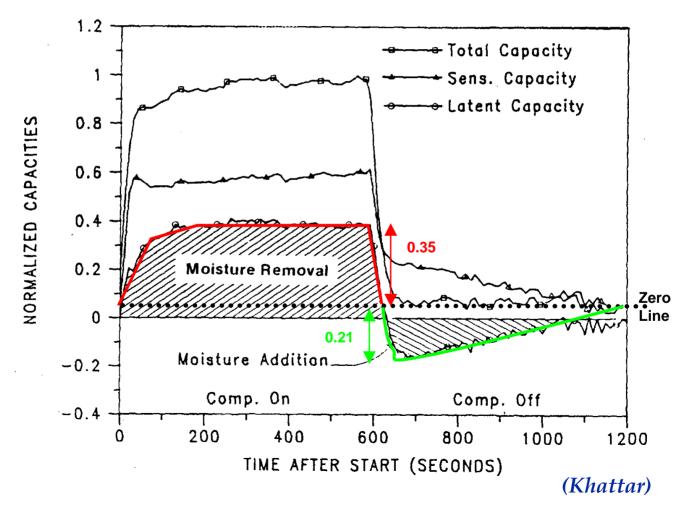
- During hot and humid weather, there is a sensible cooling load and a latent cooling load.
 - A high air flow rate will produce the highest sensible cooling efficiency.
 - A lower air flow rate will produce the highest <u>latent</u> cooling efficiency.











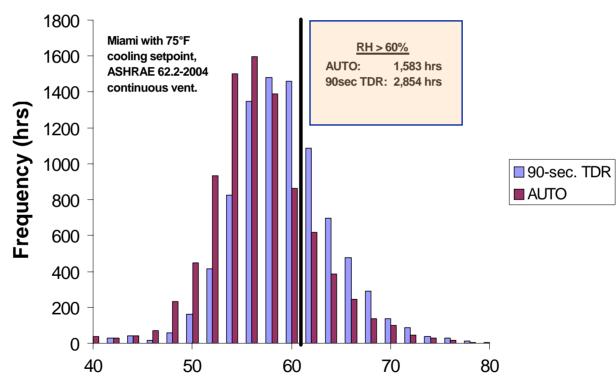




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Time Delay – Lab Results Wet-coil: 80F air inlet, 60.4F dew point			
Fan Operation	Gross EER (Btu/Wh)	Net EER considers fan heat and power (Btu/Wh)	
Auto, no overrun	11.15	9.47	
90 second overrun	11.09 (-1%)	9.10(-4%)	





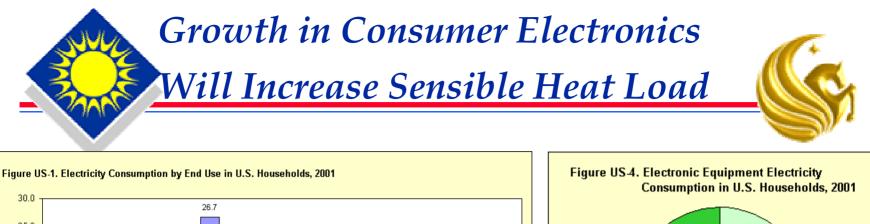
Relative Humidity (%)

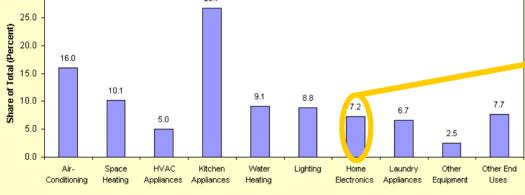
Notes: Mechanical ventilation simulated into space not coil. Used TRN-ResDH, a TRNSYS-based hour-by-hour building energy simulation tool Shirey, D.B., H.I. Henderson and R.A. Raustad. 2006. Understanding the Dehumidification Performance of Air-Conditioning Equipment at Part-Load Conditions. Final Report, FSEC-CR-1537-05



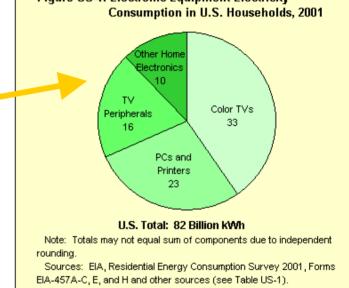


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Notes: "Share of total" is the share of total electricity consumption by U.S. households. "HVAC Appliances" consists of furnace fan, ceiling fan, dehumidifier, humidifier, and evaporative cooler (swamp cooler). "Other Equipment" consists of pool filter/pump, hot tub/spa/pool heater, waterbed heater, and well water pump. "Other End Use" includes many end uses not specifically listed. Sources: EIA, Residential Energy Consumption Survey 2001, Forms EIA-457A-C, E, and H and other sources (see Table US-1).



 EIA projects electricity consumption to grow 3.5 percent annually for color TVs and computer equipment through 2025, to more than double the level of consumption in 2003.





- More efficient windows or other envelope measures can reduce sensible loads and save energy, with slight increases in RH in typical single-family homes
- Duct leakage, time delay relay and fan speed settings of HVAC equipment can have moderate to large effect on ability of equipment to remove moisture
- Fan set to continuous "on" mode greatly reduces moisture removal
- Plug loads (sensible heat) are increasing and may counter some reductions in sensible heat from envelope improvements